CONTROL I

ELEN3016

System Modelling

(Lecture 3)

Overview

- First Things First!
- More Examples of Mechanical Systems
- Tutorial Exercises & Homework

Next Attraction!

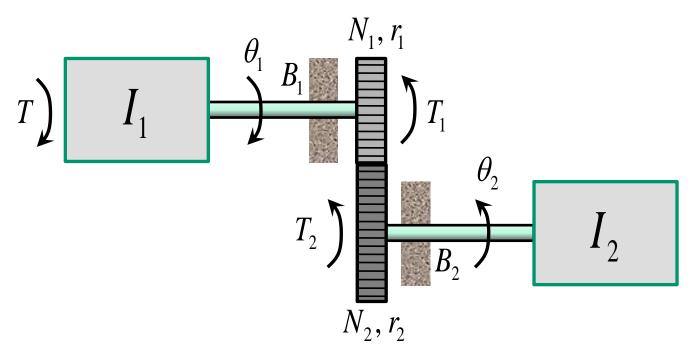
First Things First!

- Tut & Lecture Swap
 - Lecture on Wednesdays & Tut on Thursdays.
- Consultation
 - Wednesdays 8:30 10:00 AM
- Lab Matters
 - The lab brief will be available early next week.
- Lecture Notes

http://dept.ee.wits.ac.za/~vanwyk/ELEN3016_2015/

(Similar to Burns, Example 2.3)

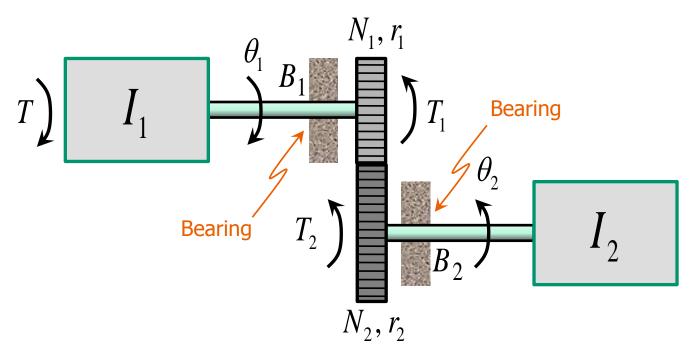
- Input: Torque, T(t) Output: Angle, $\theta_2(t)$



- Internal reaction torques: $T_1(t) = r_1 X(t)$, $T_2(t) = r_2 X(t)$
- Internal reaction force: X(t) (See to Burns, Example 2.3)

(Similar to Burns, Example 2.3)

- Input: Torque, T(t) Output: Angle, $\theta_2(t)$



- Internal reaction torques: $T_1(t) = r_1 X(t)$, $T_2(t) = r_2 X(t)$
- Internal reaction force: X(t) (See to Burns, Example 2.3)

Taper Roller Bearing



For bearing terminology visit: http://www.rbcbearings.com/tapered/components.htm

Ball Bearing



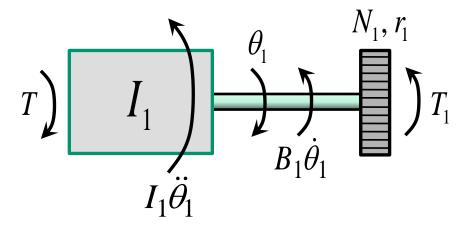
(Similar to Burns, Example 2.3)

- Simplifying assumptions:
 - We assume the gearbox to be ideal:
 - \circ The inertia of each gear-shaft assembly is small compared to the inertias I_1 and I_2
 - No friction present
 - No backlash present
- Model equations:

$$\frac{r_1}{r_2} = \frac{N_1}{N_2} = \frac{\theta_2}{\theta_1} = \frac{\dot{\theta}_2}{\dot{\theta}_1} = \frac{\ddot{\theta}_2}{\dot{\theta}_1}$$

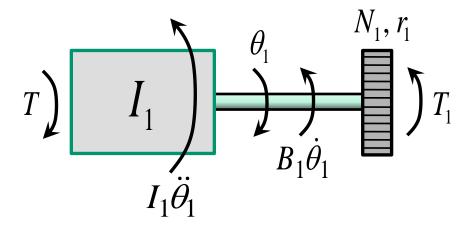
(Similar to Burns, Example 2.3)

- Free-body diagrams for inertia I_1 :



(Similar to Burns, Example 2.3)

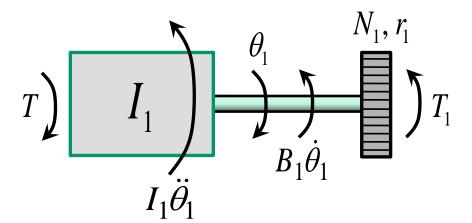
- Free-body diagrams for inertia I_1 :



Newton's 2nd law: $T(t) - I_1 \ddot{\theta}_1(t) - B_1 \dot{\theta}_1(t) - T_1(t) = 0$

(Similar to Burns, Example 2.3)

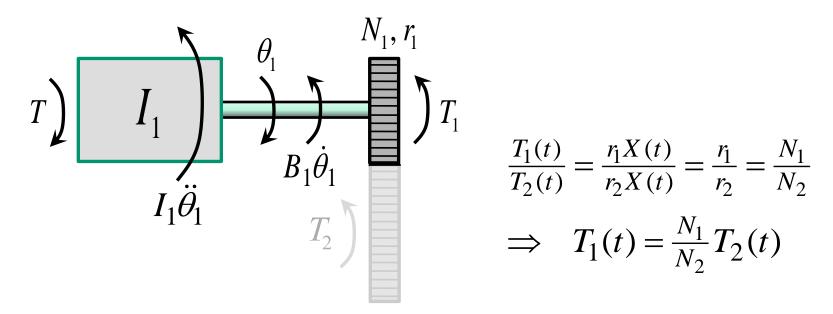
- Free-body diagrams for inertia I_1 :



Newton's 2nd law: $T(t) - I_1 \ddot{\theta}_1(t) - B_1 \dot{\theta}_1(t) - T_1(t) = 0$ $\Rightarrow T(t) - I_1 \ddot{\theta}_1(t) - B_1 \dot{\theta}_1(t) = T_1(t)$

(Similar to Burns, Example 2.3)

- Free-body diagrams for inertia I_1 :

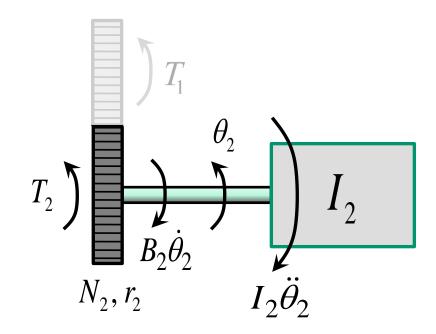


Newton's 2nd law:
$$T(t) - I_1 \dot{\theta}_1(t) - B_1 \dot{\theta}_1(t) - T_1(t) = 0$$

$$\Rightarrow T(t) - I_1 \ddot{\theta}_1(t) - B_1 \dot{\theta}_1(t) = T_1(t)$$

(Similar to Burns, Example 2.3)

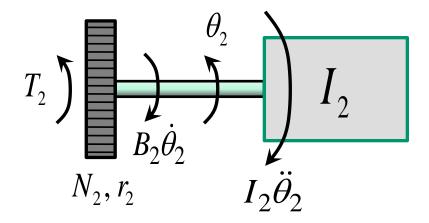
- Free-body diagrams for inertia I_2 :



(Similar to Burns, Example 2.3)

- Free-body diagrams for inertia I_2 :

Newton's 2nd law: $T_2(t) - I_2 \ddot{\theta}_2(t) - B_2 \dot{\theta}_2(t) = 0$



(Similar to Burns, Example 2.3)

- Free-body diagrams for inertia I_2 :

Newton's 2nd law: $T_2(t) - I_2\ddot{\theta}_2(t) - B_2\dot{\theta}_2(t) = 0$ $\Rightarrow I_2\ddot{\theta}_2(t) + B_2\dot{\theta}_2(t) = T_2(t)$ T_2 $R_2\dot{\theta}_2$ $R_2\dot{\theta}_2$ R_2, r_2 $R_2\ddot{\theta}_2$

(Similar to Burns, Example 2.3)

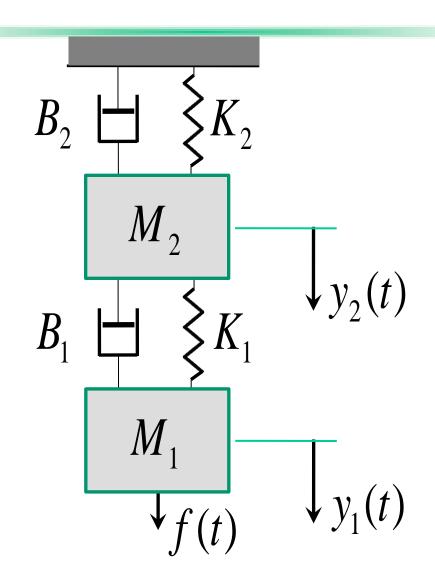
– Recalling that $\theta_2(t) = \frac{N_1}{N_2} \theta_1(t)$ and combining with the above two equations yields

$$\left(I_{1} + \left(\frac{N_{1}}{N_{2}}\right)^{2} I_{2}\right) \ddot{\theta}_{1}(t) + \left(B_{1} + \left(\frac{N_{1}}{N_{2}}\right)^{2} B_{2}\right) \dot{\theta}_{1}(t) = T(t)$$

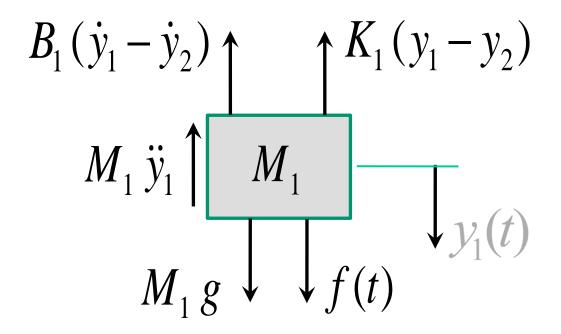
or simply

$$\widetilde{I}_1 \ddot{\theta}_1(t) + \widetilde{B}_1 \dot{\theta}_1(t) = T(t)$$
.

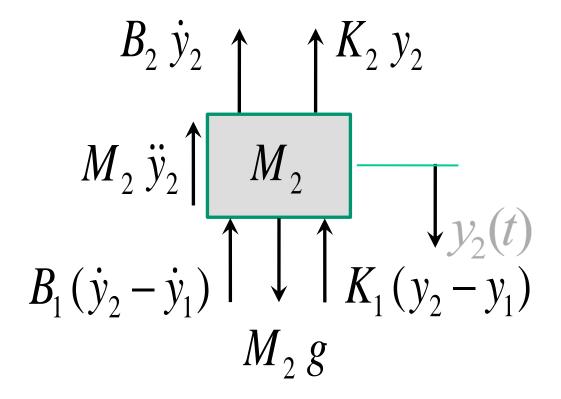
- Parameters: $N_1, N_2, I_1, I_2, B_1, B_2$



Free-body diagram for Mass 1:



Free-body diagram for Mass 2:



Analysis – NII for Mass 1:

$$-M_1\ddot{y}_1 - B_1(\dot{y}_1 - \dot{y}_2) - K_1(y_1 - y_2) + M_1g + f(t) = 0$$

$$M_1\ddot{y}_1 + B_1\dot{y}_1 + K_1y_1 - B_1\dot{y}_2 - K_1y_2 = M_1g + f(t)$$

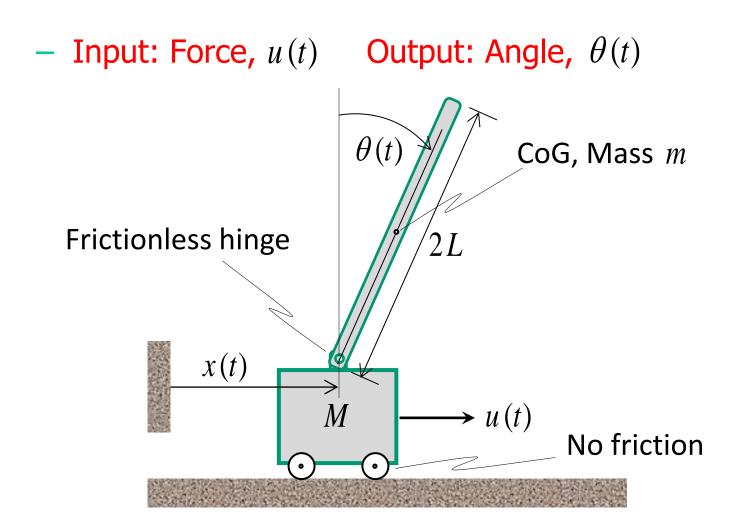
Analysis – NII for Mass 2:

$$-M_{2}\ddot{y}_{2} - B_{2}\dot{y}_{2} - K_{2}y_{2} - B_{1}(\dot{y}_{2} - \dot{y}_{1})$$

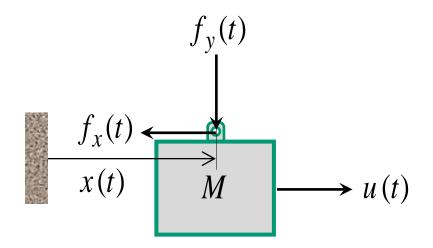
$$-K_{1}(y_{2} - y_{1}) + M_{2}g = 0$$

$$-B_{1}\dot{y}_{1} - K_{1}y_{1} + M_{2}\ddot{y}_{2} + (B_{1} + B_{2})\dot{y}_{2} + (K_{1} + K_{2})y_{2} = M_{2}g$$

(Similar to Burns, Example 10.3)



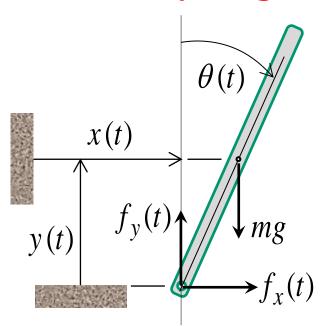
Free-body diagram for the car:



Newton's 2^{nd} law, x-direction:

$$M \ddot{x} = u - f_x$$

Free-body diagram for the pendulum:



$$I = \frac{1}{3} m L^2$$

NII of CoG in x-direction:

$$m\frac{d^2}{dt^2}(x+L\sin\theta)=f_x$$

NII of CoG in *y*-direction:

$$m\frac{d^2}{dt^2}(L\cos\theta) = f_y - mg$$

NII for rotation about CoG:

$$I \frac{d^2\theta}{dt^2} = f_y L \sin \theta - f_x L \cos \theta$$

 Eliminating the reaction forces amongst these three equations yield two equations:

$$(m+M)\ddot{x} + \frac{1}{2}mL\cos\theta\ddot{\theta} - \frac{1}{2}mL\sin\theta\dot{\theta}^2 = u(t)$$
$$\frac{1}{2}mL\cos\theta\ddot{x} + \frac{1}{3}mL^2\ddot{\theta} - \frac{1}{2}mgL\sin\theta = 0$$

 Eliminating the reaction forces amongst these three equations yield two equations:

$$(m+M)\ddot{x} + \frac{1}{2}mL\cos\theta\ddot{\theta} - \frac{1}{2}mL\sin\theta\dot{\theta}^2 = u(t)$$
$$\frac{1}{2}mL\cos\theta\ddot{x} + \frac{1}{3}mL^2\ddot{\theta} - \frac{1}{2}mgL\sin\theta = 0$$

... perform this step now.

Tutorial Exercises & Homework

Tutorial Exercises

- Inverted pendulum with linear friction B between cart and floor and torsional friction b in the hinge.
- A mechanical assembly (below) consists of two masses: a block of mass M_1 and a ball with mass M_2 , radius R_2 and moment of inertia I_2 about the centre of mass, rolling freely along the block M_1 without slipping. The block is connected to the wall by a spring K. A force f(t) is applied to the block. The frictional damping constant between the table and the block is B.

Derive the equations describing the dynamics of this system.

Tutorial Exercises & Homework

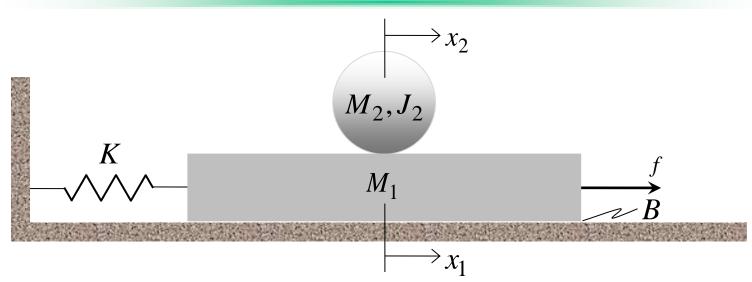


Figure: Mechanical assembly.

- Homework
 - None

Conclusion

- Example Rotational Mechanical System
- Example Translational Mechanical System
- Tutorial Exercises & Homework
- Notation/Conventions vary from Book to Book

Next Attraction! - Miss It & You'll Miss Out!

- Electromechanical Energy Conversion
- Example Brushed PM DC Motor

Thank you! **Any Questions?**